

# MECHANICS OF MATERIALS

CHAPTER

# 2

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## Stress and Strain – Axial Loading

From: Rezaei, Chapter 2, Part 1

## Stress & Strain: Axial Loading

- Suitability of a structure or machine may depend on the deformations in the structure as well as the stresses induced under loading. Statics analyses alone are not sufficient.
- Considering structures as deformable allows determination of member forces and reactions which are statically indeterminate.
- Determination of the stress distribution within a member also requires consideration of deformations in the member.
- Chapter 2 is concerned with deformation of a structural member under axial loading. Later chapters will deal with torsional and pure bending loads.

## Normal Strain

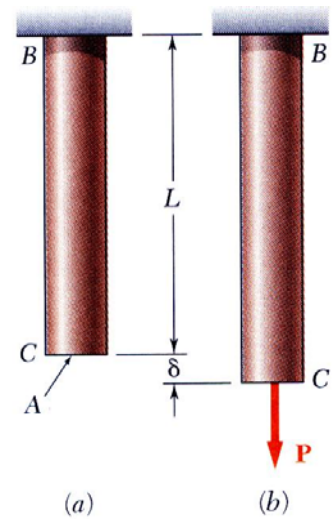


Fig. 2.1

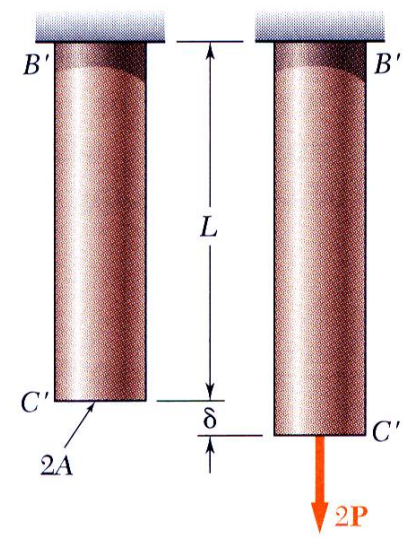


Fig. 2.3

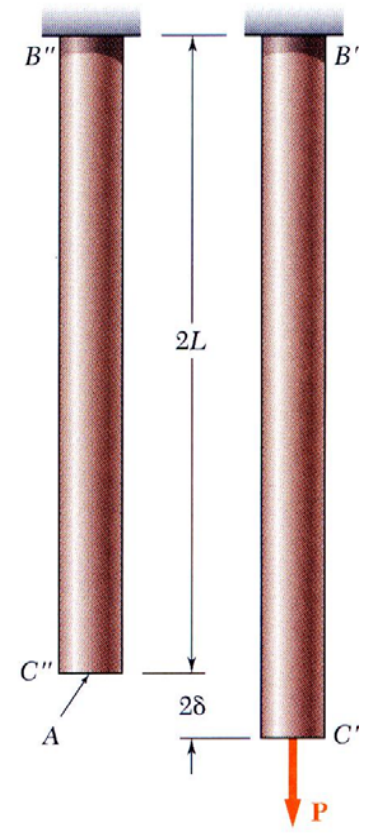


Fig. 2.4

$$\sigma = \frac{P}{A} = \text{stress}$$

$$\epsilon = \frac{\delta}{L} = \text{normal strain}$$

$$\sigma = \frac{2P}{2A} = \frac{P}{A}$$

$$\epsilon = \frac{\delta}{L}$$

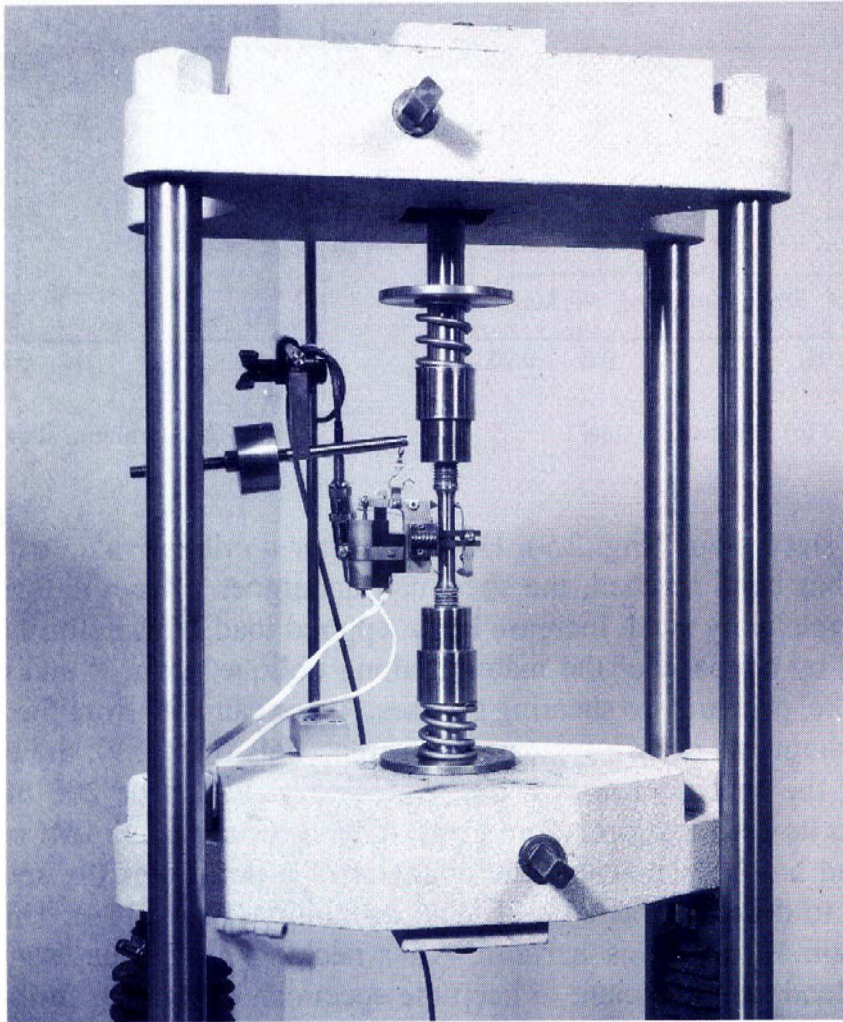
$$\sigma = \frac{P}{A}$$

$$\epsilon = \frac{2\delta}{2L} = \frac{\delta}{L}$$

From: Rezaei, Chapter 2, Part 1

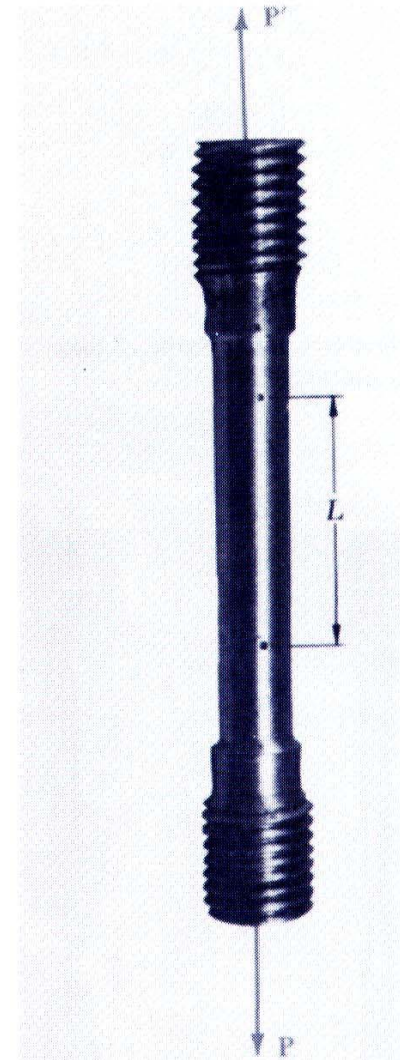


## Stress-Strain Test



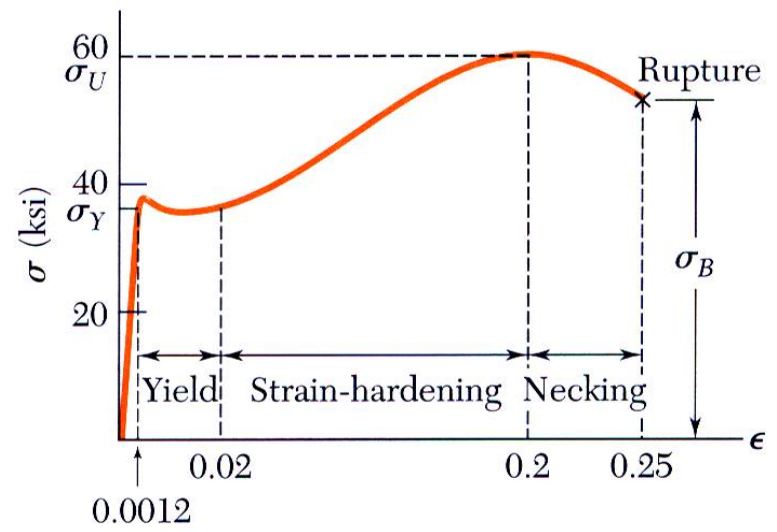
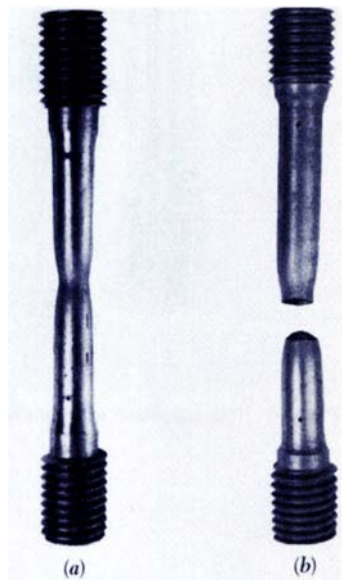
**Fig. 2.7** This machine is used to test tensile test specimens, such as those shown in this chapter.

From: Rezaei, Chapter 2, Part 1

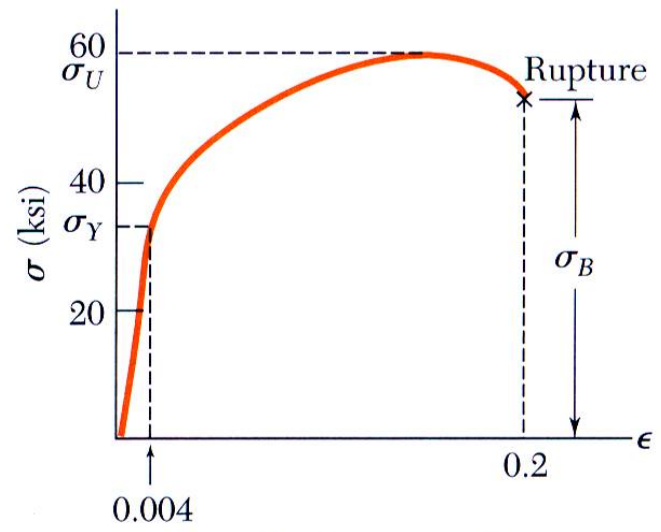


**Fig. 2.8** Test specimen with tensile load.

## Stress-Strain Diagram: Ductile Materials

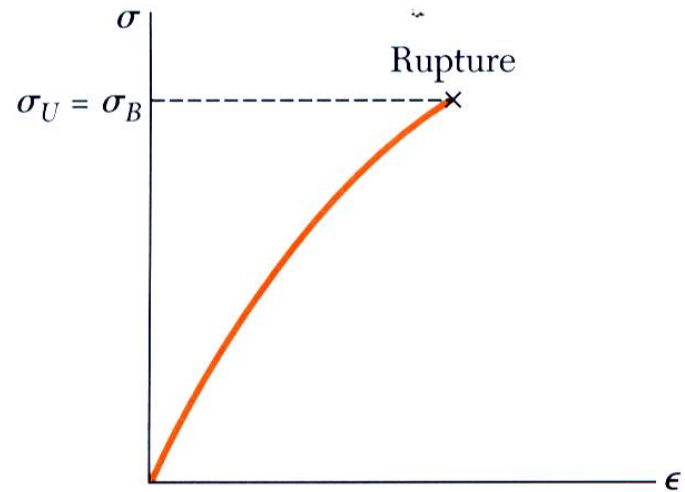
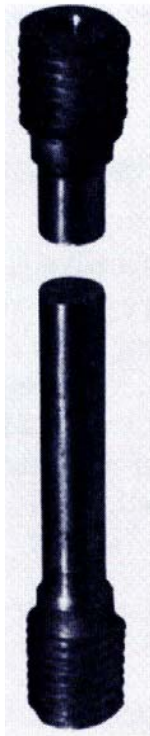


(a) Low-carbon steel



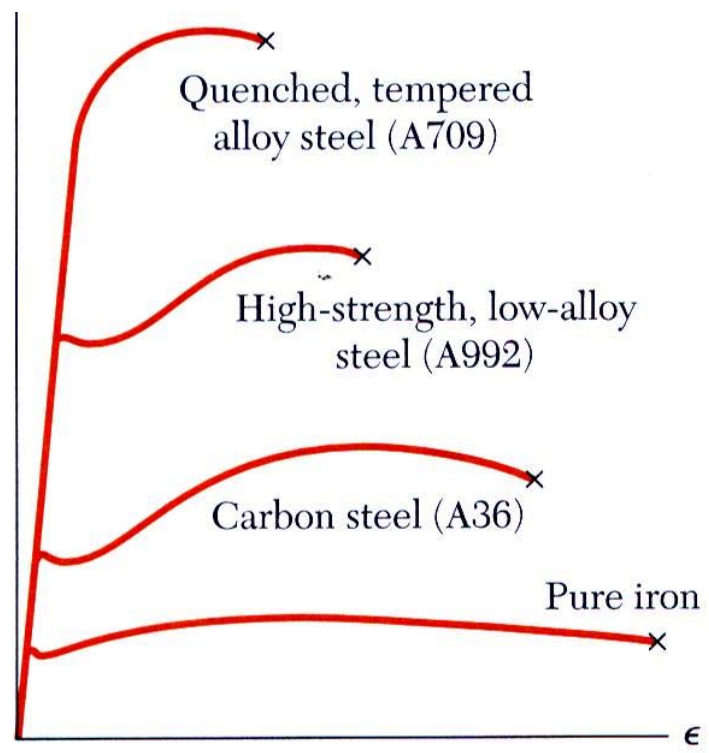
(b) Aluminum alloy

# Stress-Strain Diagram: Brittle Materials



**Fig. 2.11** Stress-strain diagram for a typical brittle material.

## Hooke's Law: Modulus of Elasticity



- Below the yield stress
  - $\sigma = E\epsilon$
  - $E =$  Youngs Modulus or Modulus of Elasticity
- Strength is affected by alloying, heat treating, and manufacturing process but stiffness (Modulus of Elasticity) is not.

**Fig. 2.16** Stress-strain diagrams for iron and different grades of steel.

## Elastic vs. Plastic Behavior

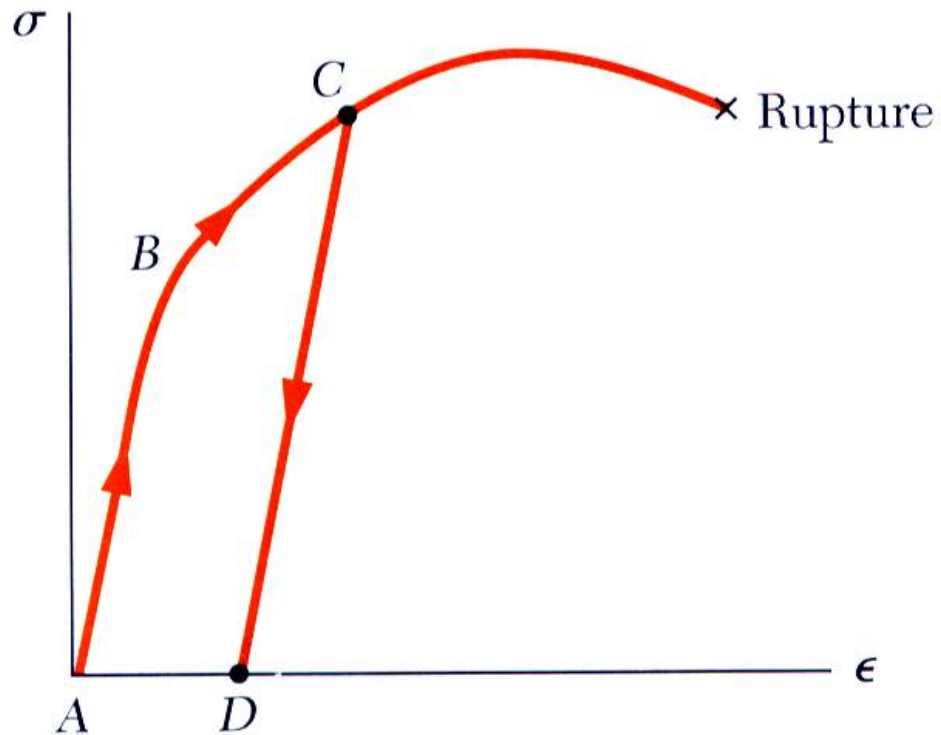


Fig. 2.18

- If the strain disappears when the stress is removed, the material is said to behave *elastically*.
- The largest stress for which this occurs is called the *elastic limit*.
- When the strain does not return to zero after the stress is removed, the material is said to behave *plastically*.



## Deformations Under Axial Loading

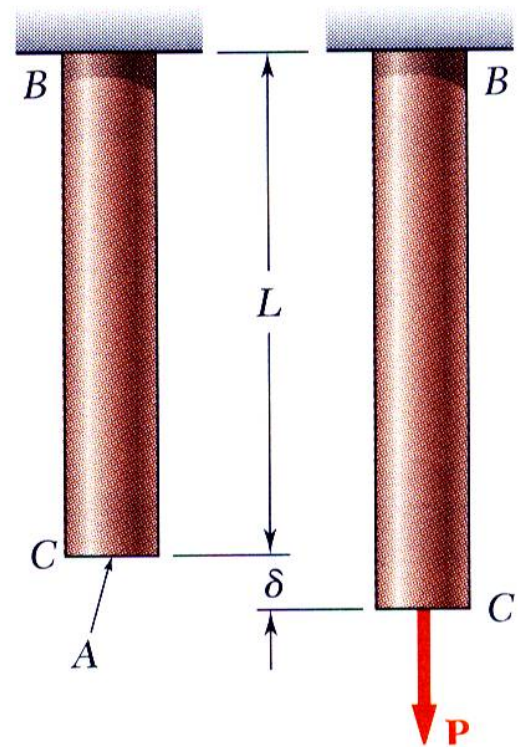


Fig. 2.22

- From Hooke's Law:

$$\sigma = E\varepsilon \quad \varepsilon = \frac{\sigma}{E} = \frac{P}{AE}$$

- From the definition of strain:

$$\varepsilon = \frac{\delta}{L}$$

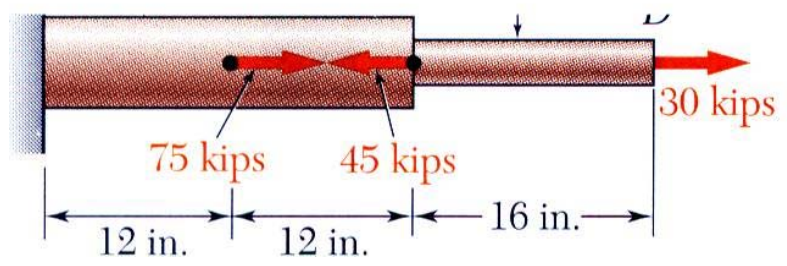
- Equating and solving for the deformation,

$$\delta = \frac{PL}{AE}$$

- With variations in loading, cross-section or material properties,

$$\delta = \sum_i \frac{P_i L_i}{A_i E_i}$$

## Example 2.01



$$E = 29 \times 10^6 \text{ psi}$$

$$D = 1.07 \text{ in.} \quad d = 0.618 \text{ in.}$$

Determine the deformation of the steel rod shown under the given loads.

### SOLUTION:

- Divide the rod into components at the load application points.
- Apply a free-body analysis on each component to determine the internal force
- Evaluate the total of the component deflections.

## SOLUTION:

- Divide the rod into three components:

- Apply free-body analysis to each component to determine internal forces,

$$P_1 = 60 \times 10^3 \text{ lb}$$

$$P_2 = -15 \times 10^3 \text{ lb}$$

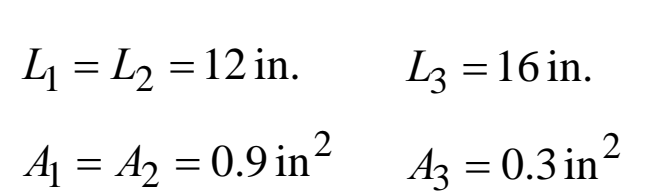
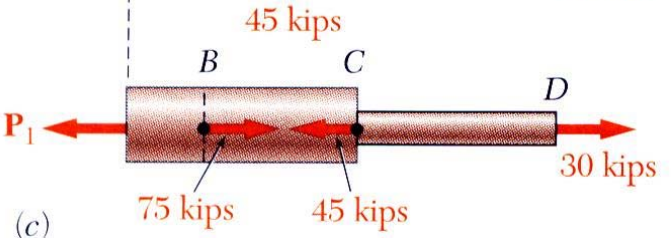
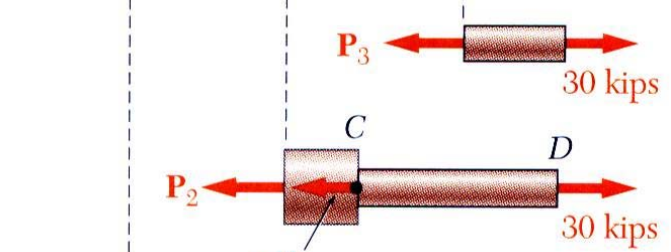
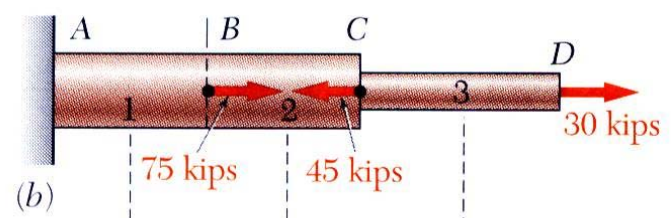
$$P_3 = 30 \times 10^3 \text{ lb}$$

- Evaluate total deflection,

$$\delta = \sum_i \frac{P_i L_i}{A_i E_i} = \frac{1}{E} \left( \frac{P_1 L_1}{A_1} + \frac{P_2 L_2}{A_2} + \frac{P_3 L_3}{A_3} \right)$$

$$= \frac{1}{29 \times 10^6} \left[ \frac{(60 \times 10^3) 12}{0.9} + \frac{(-15 \times 10^3) 12}{0.9} + \frac{(30 \times 10^3) 16}{0.3} \right]$$

$$= 75.9 \times 10^{-3} \text{ in.}$$

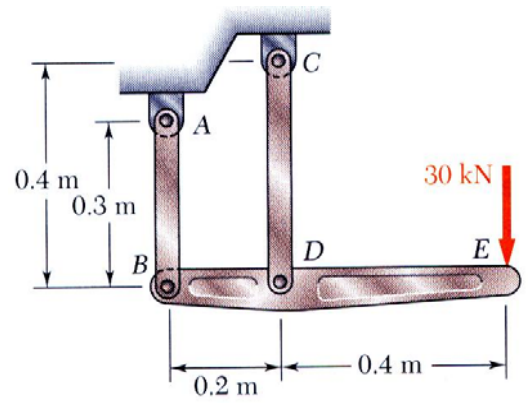


$$L_1 = L_2 = 12 \text{ in.} \quad L_3 = 16 \text{ in.}$$

$$A_1 = A_2 = 0.9 \text{ in}^2 \quad A_3 = 0.3 \text{ in}^2$$

$\delta = 75.9 \times 10^{-3} \text{ in.}$

## Sample Problem 2.1



The rigid bar  $BDE$  is supported by two links  $AB$  and  $CD$ .

Link  $AB$  is made of aluminum ( $E = 70$  GPa) and has a cross-sectional area of  $500 \text{ mm}^2$ . Link  $CD$  is made of steel ( $E = 200$  GPa) and has a cross-sectional area of ( $600 \text{ mm}^2$ ).

For the 30-kN force shown, determine the deflection a) of  $B$ , b) of  $D$ , and c) of  $E$ .

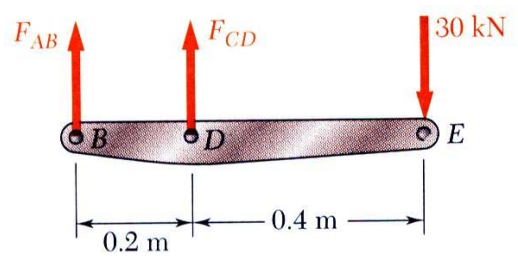
### SOLUTION:

- Apply a free-body analysis to the bar  $BDE$  to find the forces exerted by links  $AB$  and  $CD$ .
- Evaluate the deformation of links  $AB$  and  $CD$  or the displacements of  $B$  and  $D$ .
- Work out the geometry to find the deflection at  $E$  given the deflections at  $B$  and  $D$ .

## Sample Problem 2.1

SOLUTION:

Free body: Bar *BDE*



$$+\curvearrowright \sum M_B = 0$$

$$0 = -(30 \text{ kN} \times 0.6 \text{ m}) + F_{CD} \times 0.2 \text{ m}$$

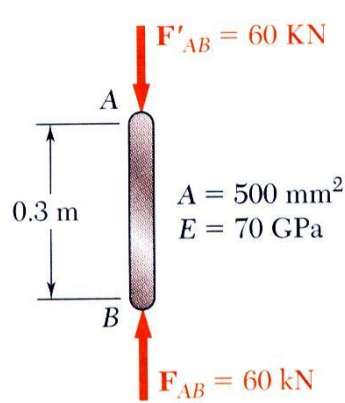
$$F_{CD} = +90 \text{ kN } \textit{tension}$$

$$+\curvearrowright \sum M_D = 0$$

$$0 = -(30 \text{ kN} \times 0.4 \text{ m}) - F_{AB} \times 0.2 \text{ m}$$

$$F_{AB} = -60 \text{ kN } \textit{compression}$$

Displacement of *B*:



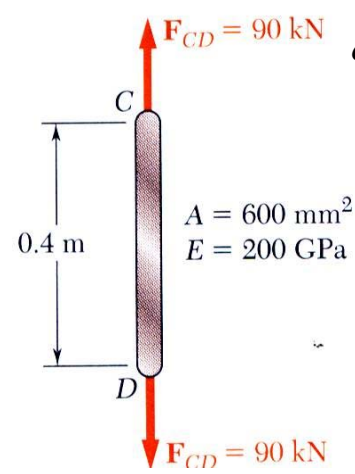
$$\delta_B = \frac{PL}{AE}$$

$$= \frac{(-60 \times 10^3 \text{ N})(0.3 \text{ m})}{(500 \times 10^{-6} \text{ m}^2)(70 \times 10^9 \text{ Pa})}$$

$$= -514 \times 10^{-6} \text{ m}$$

$$\delta_B = 0.514 \text{ mm } \uparrow$$

Displacement of *D*:



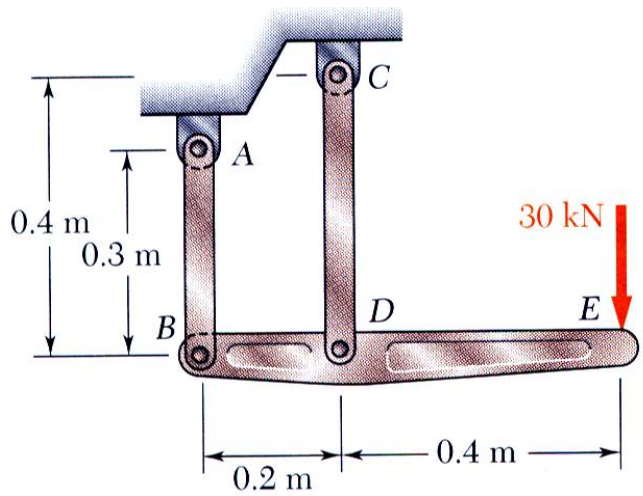
$$\delta_D = \frac{PL}{AE}$$

$$= \frac{(90 \times 10^3 \text{ N})(0.4 \text{ m})}{(600 \times 10^{-6} \text{ m}^2)(200 \times 10^9 \text{ Pa})}$$

$$= 300 \times 10^{-6} \text{ m}$$

$$\delta_D = 0.300 \text{ mm } \downarrow$$

## Sample Problem 2.1

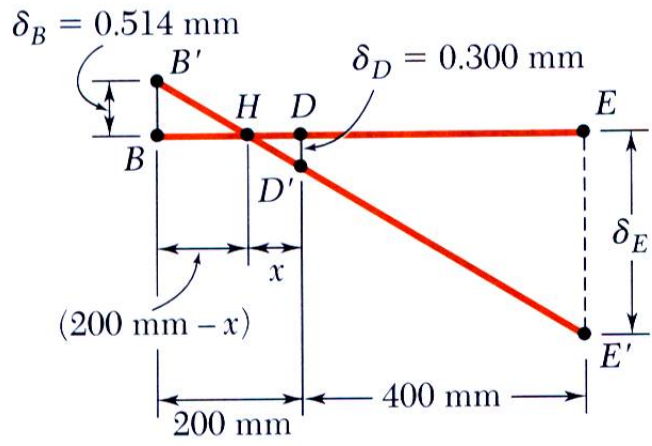


Displacement of D:

$$\frac{BB'}{DD'} = \frac{BH}{HD}$$

$$\frac{0.514 \text{ mm}}{0.300 \text{ mm}} = \frac{(200 \text{ mm}) - x}{x}$$

$$x = 73.7 \text{ mm}$$



$$\frac{EE'}{DD'} = \frac{HE}{HD}$$

$$\frac{\delta_E}{0.300 \text{ mm}} = \frac{(400 + 73.7) \text{ mm}}{73.7 \text{ mm}}$$

$$\delta_E = 1.928 \text{ mm}$$

$\delta_E = 1.928 \text{ mm} \downarrow$